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High solar flux heating of upflow bubbling fluidized bed circulating in opaque vertical tube - 3d numerical simulation

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3D Numerical Simulation of Upflow Bubbling Fluidized Bed in Opaque Tube under High Flux Solar Heating

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May 25, 2016

Introduction

- 2 Single-tube pilot rig
- 3 Modeling approach
- 4 Numerical results
- 5 Conclusions and Perspectives

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Introduction Exp. Simulations Numerical results Conclusions

European project CSP2: Solar receiver



Main objectives:

- To propose an alternative to current Heat Transfer Fluid for concentrated solar power plants that can operate in a wide range of temperature (100°C-1,000°C) without freezing and decomposition.
- Geldart's group A particle of silicon carbide, easily fluidizable.

Introduction Exp. Simulations Numerical results Conclusions

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Introduction Exp. Simulations Numerical results Conclusions

Experimental Pilot Rig



Side photograph of the setup set at the CNRS solar furnace focus.



Introduction Exp. Simulations Numerical results Conclusions

Experimental Pilot Rig



Schematic cross-sectional view of the lab-scale solar rig.

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Particle recirculation



- $T_{cavity,inlet} > T_{DiFB}$
- Cavity inlet: $T_{close-to-wall} > T_{center}$.
- Cavity outlet: $T_{close-to-wall} < T_{center}$.

Recirculation confirmed by:

- 3D numerical simulations.
- Positron Emission Particle Tracking (PEPT) (University of Surrey).

Image: A matrix and a matrix

Experimental results

- Dense Particle Suspension can work as HTF.
- Suspension temperature up to 750 °C.
 ⇒ high efficiency thermodynamic cycles.
- $\bullet\,$ Heat transfer coefficients up to 1100 $\rm W/m^2.K.$
- 150 kWth 16-tube pilot successfully tested (multi-tube concept validated).

Reproduce process by numerical simulations:

- To obtain additional informations on the flow behavior.
- To develop a numerical simulation tool coupling hydrodynamics and heat transfer for the design and the optimization of a multi-megawatt particle solar receiver.



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Mathematical model

Eulerian n-fluid modeling approach for turbulent and polydispersed fluid-particle is implemented in NEPTUNE_CFD, unstructured parallelized CFD multiphase flow code.

- Fluid turbulence modeling: no model.
- Particle stress modeling based on granular kinetic theory $q_{
 ho}^2 q_{f
 ho}.$
- Convective/diffusive heat transfer between the gas and dispersed phase and inter-particle radiative transfer (Rosseland)

Numerical parameters



Computations run on 140 cores in parallel. 2 steps (more than 1 month):

- Transitory $\frac{dm_{solids}}{dt} \neq 0$, $T_{predicted,outlet} \neq T_{measured,outlet}$.
- Statistics for comparisons during 150 secondes

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Numerical parameters



Solar receiver hydrodynamics

(日)

Introduction

- 2 Single-tube pilot rig
- 3 Modeling approach

4 Numerical results

5 Conclusions and Perspectives

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Comparisons with experiments

- Problem: Recirculation overestimated. \Downarrow
- Temperature overestimated at cavity inlet.
- Temperature underestimated at cavity outlet.



General comparisons

Parameter	Medium solid flux	High solid flux
gas pressure drop	-4.6%	+2.6%
T_p cavity inlet	+26%	+9%
T_p cavity outlet	-2.7%	+0.2%

• The errors decrease when the solid mass flux increases (decrease on downward solid mass flow).

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Solid volume fraction



Vertical net solid mass flux



- Gp wall < 0 and Gp center $> 0 \Leftrightarrow$ recirculation.
- Aeration \Rightarrow recirculation increases (air velocity increases).
- T increases \Rightarrow recirculation increases.



Introduction

- 2 Single-tube pilot rig
- 3 Modeling approach

4 Numerical results



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Conclusions

- Dense Particle Suspension upward flow in tube simulated with NEPTUNE_CFD.
- Particle recirculation reproduced but overestimated.
- Accurate linear pressure drop.
- Influence of aeration and temperature on the vertical and horizontal solid velocities and on the time variance of the solids velocities (not presented).
- Heat transfer from the wall to the center governed by particles' collective movement.

Perspectives:

- Better account for the particle-particle friction, for the particle size distribution and highly irregular shape.
- Simulate non-uniform heat flux.
- Effect of higher tube and greater temperature increase.